

NONLINEARITY OF DEGENERATELY DOPED FLEXURAL MODE SILICON MICROMECHANICAL RESONATORS

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ABSTRACT

In this paper, we present an experimental study of nonlinearities in degenerately doped flexural-mode silicon MEMS resonators. Two geometries (single and double-anchored) and two wafer orientations ($\langle 100 \rangle$ and $\langle 111 \rangle$) are used to analyze the elastic effects of p-type (1.31×10^{16} to $1.61 \times 10^{15} \text{ cm}^{-3}$) and n-type (1.264×10^{15} to $2.64 \times 10^{14} \text{ cm}^{-3}$) degenerately doped double-ended tuning forks. We employ closed-loop frequency sweeps to characterize the amplitude-frequency response of each system. Experimental results show that geometric nonlinearities and electrostatic nonlinearities dominate material and crystalline orientation-dependent nonlinearities in the flexural mode. These results present an initial exploration into characterizing nonlinear mechanical behavior of resonators operating in lower frequency bending modes.

KEYWORDS

Flexural mode resonator; nonlinear oscillators; degenerate doping; geometric nonlinearity

INTRODUCTION

Electrostatically actuated microelectromechanical (MEMS) oscillators have become a successful alternative to quartz crystal as they provide benefits such as compatibility with on-chip CMOS circuitry and size reduction for commercial applications [1]. The reduced device size and mass, however, comes with the drawback of lower power handling capability and smaller output capacitances. As the resonator is driven to larger amplitudes to overcome the reduced signal-to-noise ratios, devices can enter the nonlinear region where the amplitude-dependent higher order terms can dominate system behavior [2]. Duffing nonlinearities and nonlinear dissipation are examples of parameters that become more significant.

Conservative nonlinearities can be broadly categorized as electrostatic nonlinearities and mechanical (geometric, material, orientation) nonlinearities [3]. Previous studies have characterized the nonlinearities of doped bulk mode resonators [4], including the material nonlinear limits of Lamé resonators [5] nonlinearity in n-type doped silicon resonators [6], and analytical models of the nonlinear Young's modulus [7]. Work on the flexural mode resonators has been limited because of the generally high feedthrough effects at larger input voltages. This work focuses on double-ended tuning fork (DETF) resonators that have been characterized as temperature references and well-studied for their dissipative nonlinearities [8]. The potential reduction in the frequency-temperature coefficient for silicon MEMS makes degenerately doped

silicon resonators an especially interesting device family of study. Figure 1 shows the two kinds of DETFs utilized in our study: single-anchored (SA) and double-anchored (DA) with the flexural mode resonant frequency at an average of 1.10 MHz for SA-DETFs and 1.12 MHz for DA-DETFs. These two devices provide a contrasting comparison since the addition of the second anchor increases the overall strain in the device, which can bring geometric stiffening nonlinearities into play. All devices in this work were fabricated in the epi-seal encapsulation process [9] [10] with varying doping levels and orientations as labeled in the following figures.

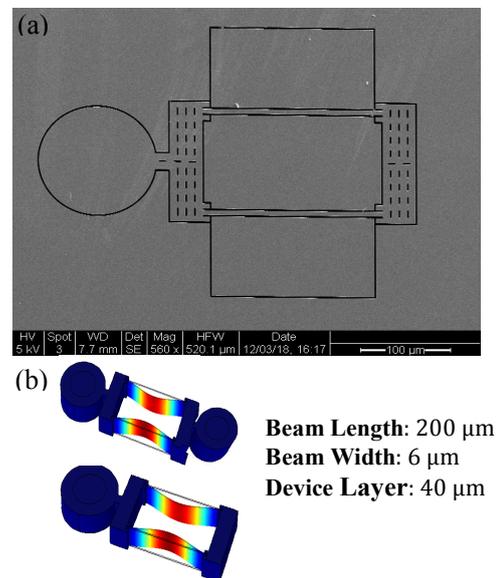


Figure 1: (a) SEM of DETF device (top-down view) and (b) The COMSOL model of a SA (bottom) and DA (top) DETF device operating in the symmetric flexural mode.

The table below summarizes the resistivity and doping concentration information for p-type and n-type dopants in on $\langle 100 \rangle$ and $\langle 111 \rangle$ wafers. This also summarizes the array of devices we present in this study.

Table 1: Resistivity and dopant concentration for p-type and n-type dopants on $\langle 100 \rangle$ and $\langle 111 \rangle$ wafers.

Dopant	Resistivity ($\text{m}\Omega - \text{cm}$)	Concentration (cm^{-3})
Boron	$\langle 100 \rangle$: 1.13	$\langle 100 \rangle$: 1.31×10^{16}
	$\langle 111 \rangle$: 8.47	$\langle 111 \rangle$: 1.61×10^{15}
Antimony	$\langle 100 \rangle$: 3.66	$\langle 100 \rangle$: 1.36×10^{15}
	$\langle 111 \rangle$: 17.0	$\langle 111 \rangle$: 2.64×10^{14}

THEORY

The dynamics of the vibrational mode of a nonlinear MEMS resonator can be modeled as a single-degree of freedom lumped mass system with an equation of motion:

$$m\ddot{x} + c\dot{x} + k_0x + k_1x^2 + k_2x^3 = F\cos(\omega t) \quad (1)$$

where x is the modal displacement, m is the lumped mass, c is the linear damping coefficient, k_0 is the linear spring constant, k_1 and k_2 represent higher order nonlinear spring constants, and F is the magnitude of the external periodic input drive to the system. The system's linear natural frequency is defined as $\omega_0 = \sqrt{k_0/m}$.

The analytical frequency response results referenced from [11] and [12] describe the change in the resonant frequency (f) as a function of the change in amplitude (a) in the presence of a given nonlinearity.

$$f \cong f_0 + \kappa a^2 \rightarrow \kappa = \Delta f/a^2 \quad (2)$$

The resulting ratio (κ) is the amplitude-frequency coefficient of the system (A - f coefficient) and it can be estimated with the first and the second order anharmonic stiffness coefficients. The magnitudes and polarities of stiffness coefficients k_0, k_1, k_2 predict the shift in the resonance peak either towards the lower frequencies ("softening") or towards the higher frequencies ("hardening").

$$\kappa = \left(\frac{3}{8} \frac{k_2}{k_0} - \frac{5}{12} \frac{k_1^2}{k_0^2} \right) f_0 \quad (3)$$

Mechanical nonlinearities are the result of either intrinsic properties of the crystal lattice or additional kinematic effects resulting from changes to the stress-strain distribution. The effect of these nonlinearities can be described by the nonlinear engineering Young's modulus for a resonator body subject to large deformations:

$$\mathbf{E} = \mathbf{E}_0 + \mathbf{E}_1\epsilon + \mathbf{E}_2\epsilon^2 \quad (4)$$

where ϵ is the engineering strain, \mathbf{E} is the effective Young's modulus, \mathbf{E}_0 is the linear Young's modulus, and \mathbf{E}_1 and \mathbf{E}_2 are the higher order correction terms due to the large strain factor. The contribution of free electrons on the second and third order elastic constants for n- and p-type silicon can be used to analytically compute the nonlinear engineering Young's modulus.

Electrostatic nonlinearities, on the other hand, arise from the variable gap capacitive transduction and are a function of the electrode and resonator configuration. The analytical model from Yang *et. al* estimate the first order electrostatic spring constant shift in resonant frequency as a function of bias voltage,

$$f_0 \cong \frac{1}{2\pi} \sqrt{\frac{1}{m} \left(k_0 - \frac{N\epsilon_0 V_b^2 A}{d_0^3} \right)} \quad (5)$$

where N is the total number of electrodes, V_b is the dc bias voltage and d_0 is the initial transduction gap size, and A is the transduction area. Equation 5 demonstrates that the

resonant frequency is a function of the bias voltage; specifically, increasing the bias voltage shifts the linear resonant frequency to lower frequencies.

CHARACTERIZATION METHOD AND EXPERIMENTAL RESULTS

The experimental setup used to perform closed loop frequency sweeps is included in Figure 2. To prepare the resonator for testing, encapsulated epi-seal devices are mounted to a printed circuit board via a chip carrier, wire bonded to the package, and placed in an environmental control chamber at 30 ± 0.1 °C for temperature control. The ac output signal from a Zurich HF2LI lock-in amplifier excites the resonator while the resonator body remains biased at a constant dc voltage of 35 V. The resonator output current is converted into a voltage using a trans-impedance amplifier (TIA). By maintain a variable-phase feedback loop on the resonator, we can track the amplitude as a function of frequency of the resonator beyond the critical bifurcation regime [13].

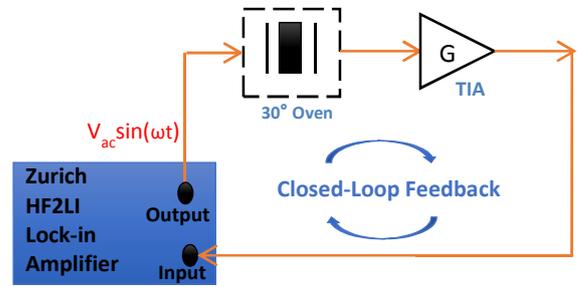


Figure 2: Experimental apparatus used for closed loop frequency sweep measurements

The resonator displacement a is estimated as a first order approximation from the output voltage of the TIA (V_{out}) scaled by the gain of the TIA (G_{tot}):

$$a \cong \frac{V_{out} d_0^2}{2\pi f_0 G_{tot} V_b \epsilon_0 A} \quad (6)$$

Figure 3 shows the closed-loop frequency response for resonators of different doping and wafer orientation. Each color represents increasing input ac drive voltage to the resonator from low drive (blue) to high drive voltages (red).

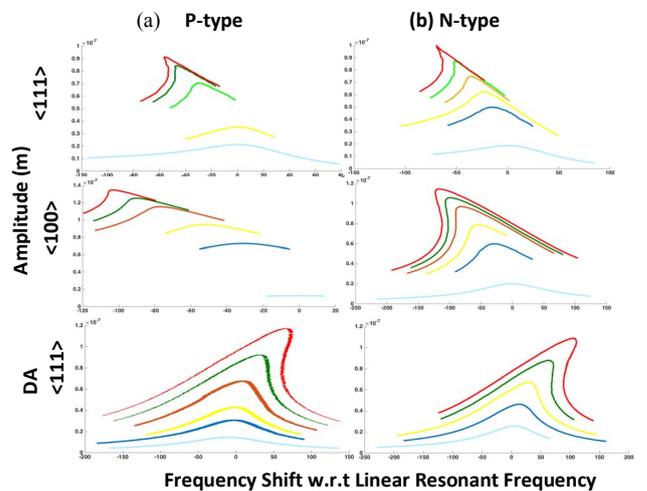


Figure 3: Closed loop frequency sweeps of (a) p-type (b) n-type wafers of two different orientations. The top two rows are sweeps of SA-DETFs and the bottom row are sweeps of DA-DETFs.

SA-DETFs on both $\langle 100 \rangle$ $\langle 111 \rangle$ wafers exhibit a softening nonlinearity regardless of doping while DA-DETFs on $\langle 111 \rangle$ wafers exhibit a hardening nonlinearity regardless of dopant-type.

From the closed-loops measurements, the value of the amplitude-frequency coefficient κ can be calculated through tracking the maximum vibrational amplitude and determining the corresponding frequency shift, measured in hertz, with respect to the linear resonant frequency. The results are then fit with Equation 2 and plotted in Figure 4.

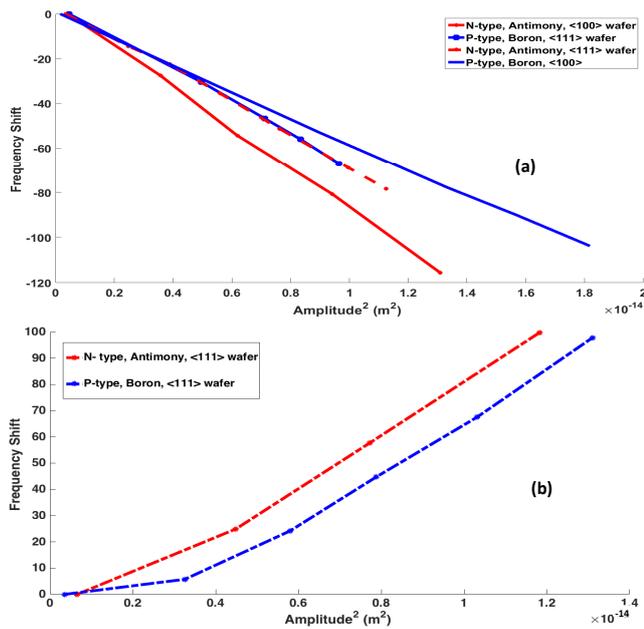


Figure 4: (a) Combined plots of the amplitude-frequency coefficients for SA DETFs of various doping and wafer orientation (b) Combined amplitude-frequency coefficients for DA DETFs of various doping. Frequency shift is measured in Hz as the difference from the linear resonant frequency.

We explore the repeatability our data set due to external factors including the variability in a DETF on the same die oriented in the 0° versus 90° direction, reticle to reticle variability, variability due to wire-bonding of the die to the printed circuit board, and variability due to die thickness. To explore the die thickness variability, a die was thinned to a total thickness of $150 \mu\text{m}$ and compared to a non-thinned die on the same wafer. Figure 5 represents the reticle to reticle variability for both SA and DA-DETFs while Figure 6 shows that repeatability is independent of external factors. In Figure 6 (b) and 6 (c) the run to run variability refers to obtaining the same set of data multiple times to understand errors due to oven temperature fluctuations, lock-in frequency drift, and any other environmental changes. These results all confirm that the SA-devices are dominated by the electrostatic softening nonlinearity while the DA-devices are dominated by the geometric hardening nonlinearity, and the materials effects are secondary.

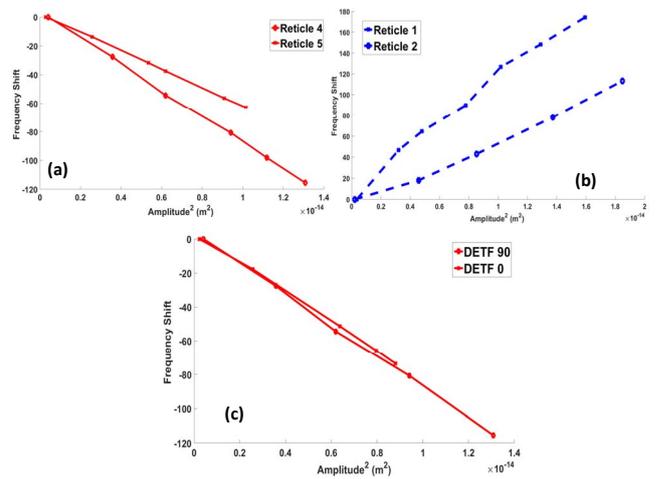


Figure 5: Reticle-to-reticle variability of the amplitude-frequency coefficient for (a) N-type, Antimony, $\langle 100 \rangle$, SA-DETF (b) P-type, $\langle 111 \rangle$ wafer, DA-DETF and (c) for a DETF on a given wafer, variability due to the 0° and 90° deg orientation.

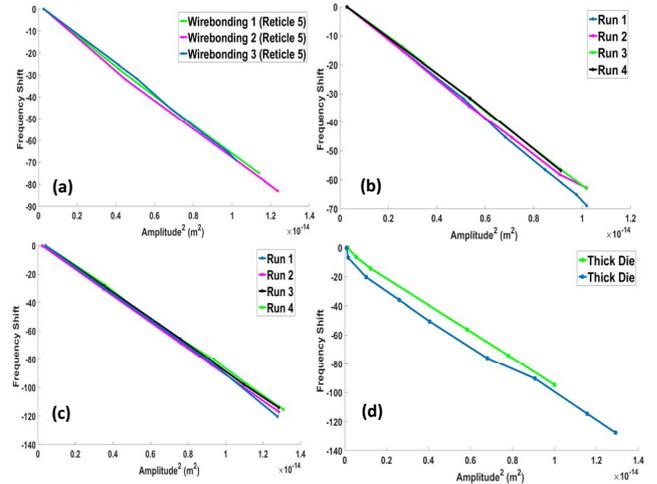


Figure 6: Experimental repeatability independent of external factors including (a) wirebonding (b) run-to-run variation on the N-type, Antimony SA-DETF on reticle 5 (c) run-to-run variation on the N-type, Antimony, SA-DETF from HDXIII on reticle 4 (d) thick vs. thinning of the die

CONCLUSIONS

Experimental results show that electrostatic and geometric nonlinearities dominate any material-dependent nonlinear effects for flexural mode resonators. We observe high consistency between data set to data set, wire-bonding, and devices on the same reticle. However, the variability between two different reticles, which is on the same order magnitude as the variability due to doping suggests that we cannot, at this time, quantify the amplitude-frequency coefficients for a specific geometry and doping. This work will continue with careful exploration of nonlinearities in other MEMS flexural resonators, such as disk gyroscopes, which operate with similar flexural modes.

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